

ANSYS CFD – SESSION 1

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Univerza v Ljubljani



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- Why meshing?
- Building blocks and terminology
- Mesh types
- Cell types
- Mesh quality
- Hands on: building CFD mesh for simulation of laminar flow in 2D U-bend channel

Why meshing?

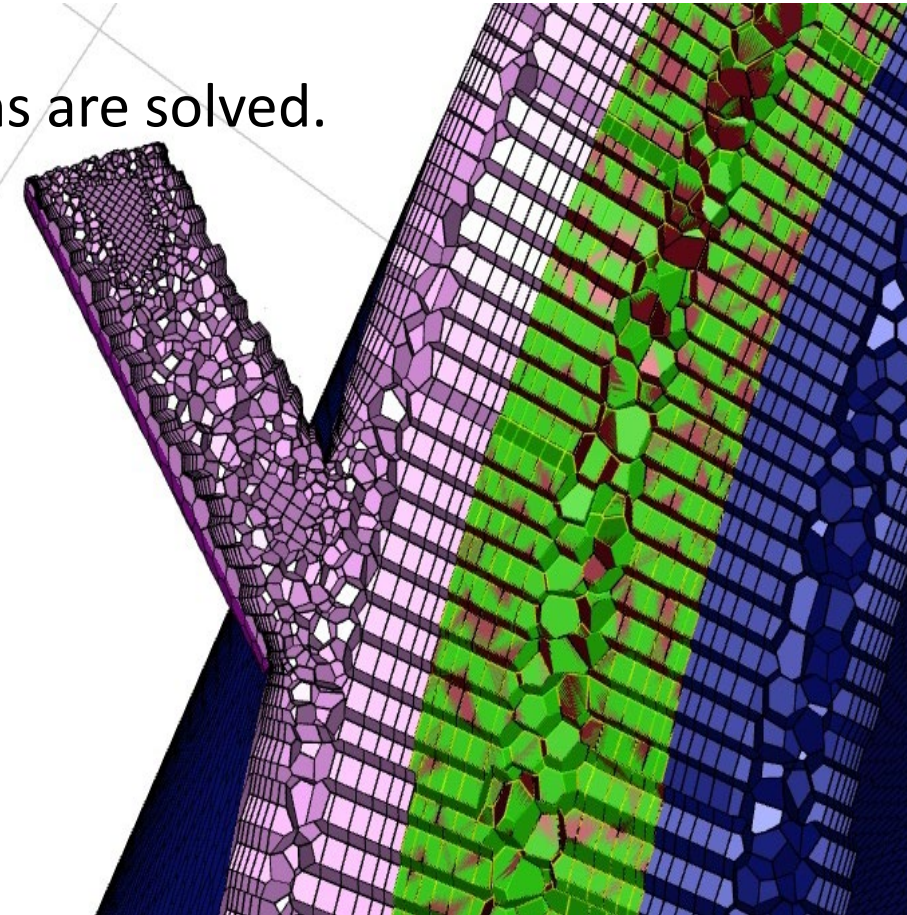
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Why do we need mesh/grid?

- Designates the “elements” on which the flow equations are solved.
- Gives discrete representation of the geometry.
- Has entities (groups of cells) where b.c.’s are applied.

The mesh has impact on:

- Rate of convergence (or even lack of convergence).
- Solution accuracy.
- CPU time required.



Building blocks

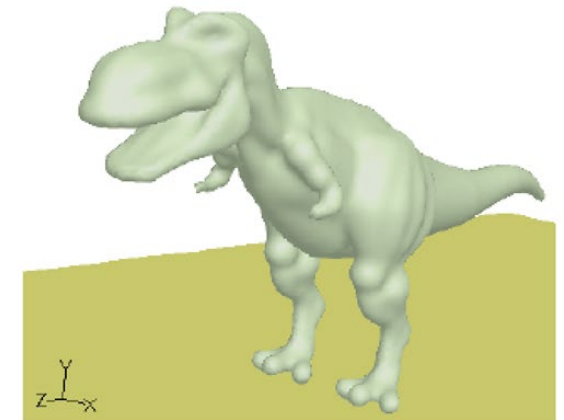
- Geometry is the starting point of any CFD study

Geometry can be:

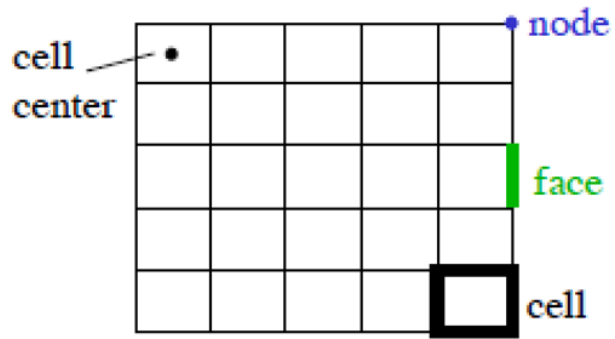
- Basic or complex
- Imported from CAD
- Created using bottom-up/top-down/hybrid approach

Is composed of:

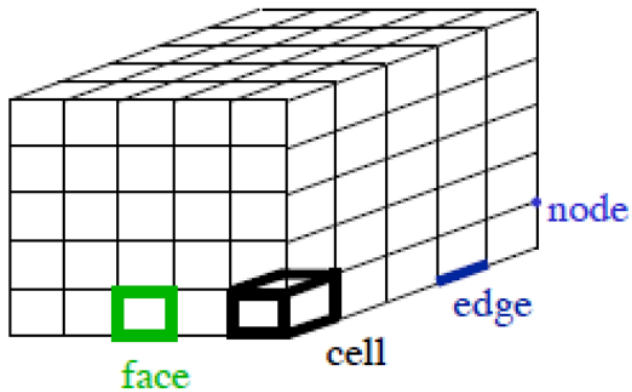
- Volumes
- Surfaces
- Curves
- points



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2D computational grid



3D computational grid

Mesh terminology

- Cell – basic volume unit of the mesh
- Node – point when sides of different cells meet
- Cell center – center of the cell
- Edge – boundary of face
- Face – boundary of cell
- Zone – group of cells, faces, nodes
- Domain – group of zones

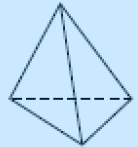
Cell types



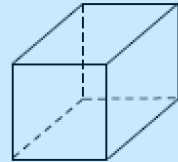
triangle
("tri")



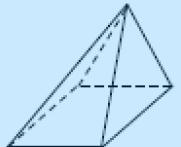
2D prism
(quadrilateral
or "quad")



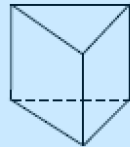
tetrahedron
("tet")



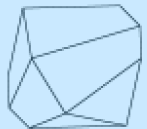
prism with
quadrilateral base
(hexahedron or "hex")



pyramid



prism with
triangular base
(wedge)



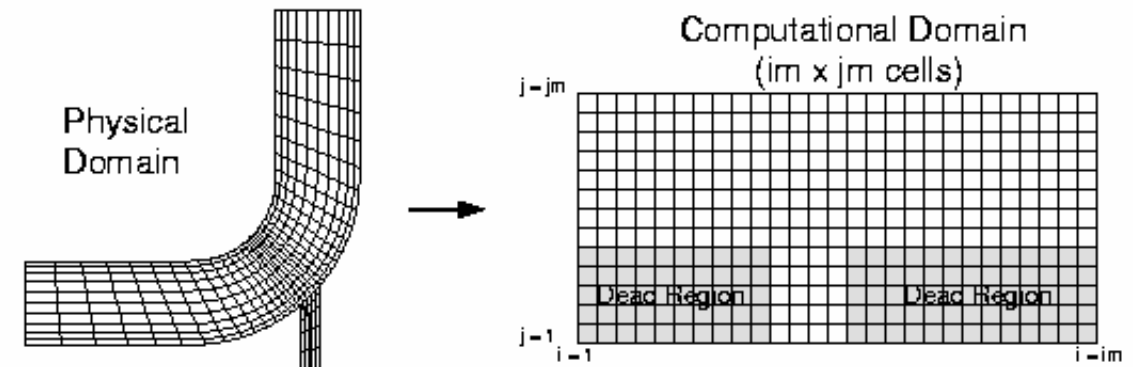
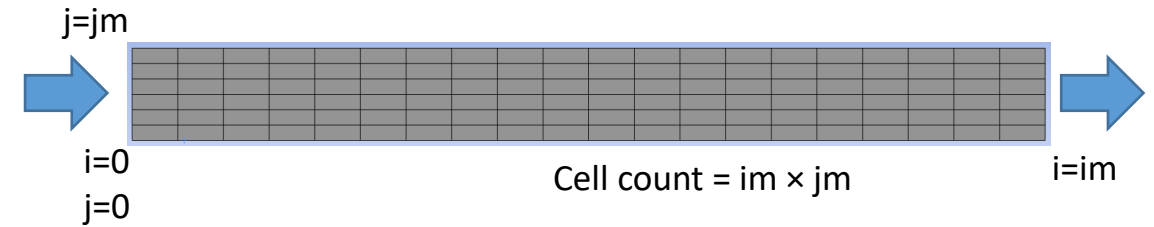
arbitrary polyhedron

Mesh types

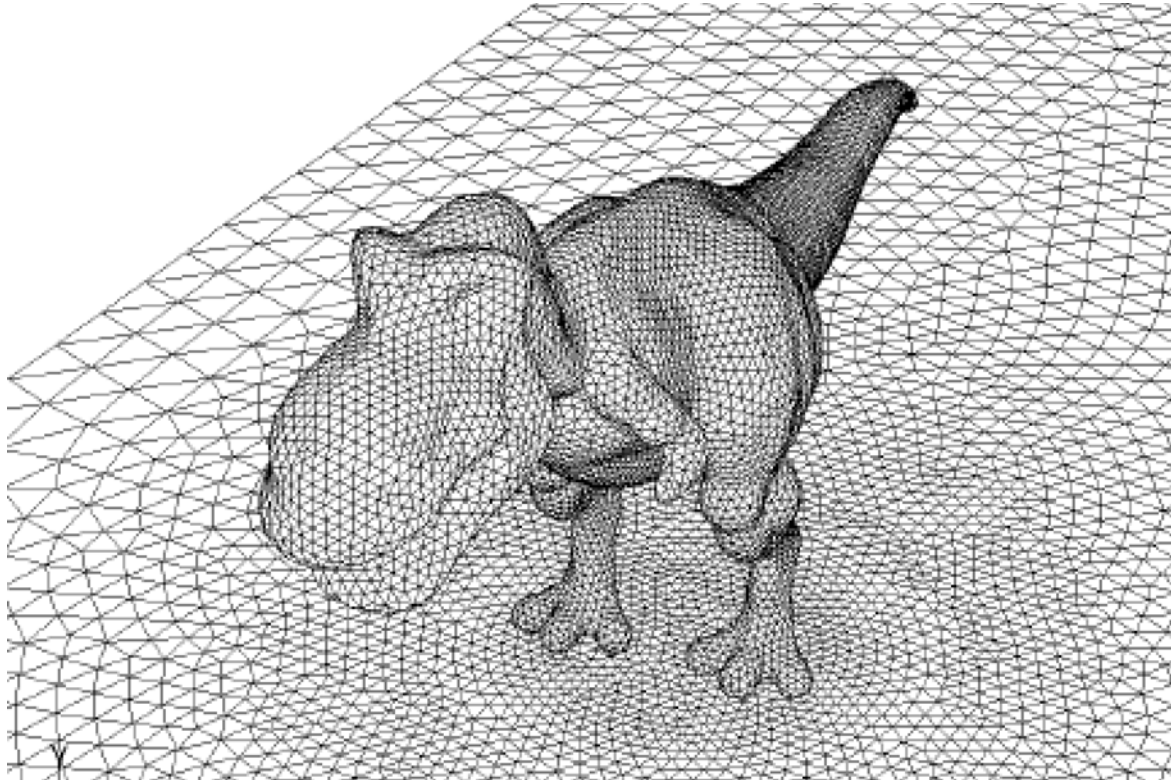
- Structured
- Unstructured
- Hybrid

Structured mesh

- The cells distribution follows the flow topology
- The cells are indexed
- Blocking analogy is used
- Structured meshes are of high quality and very good cell count/volume ratio
- Harder to do (compared to unstructured)
- Limited use for highly complex geometry



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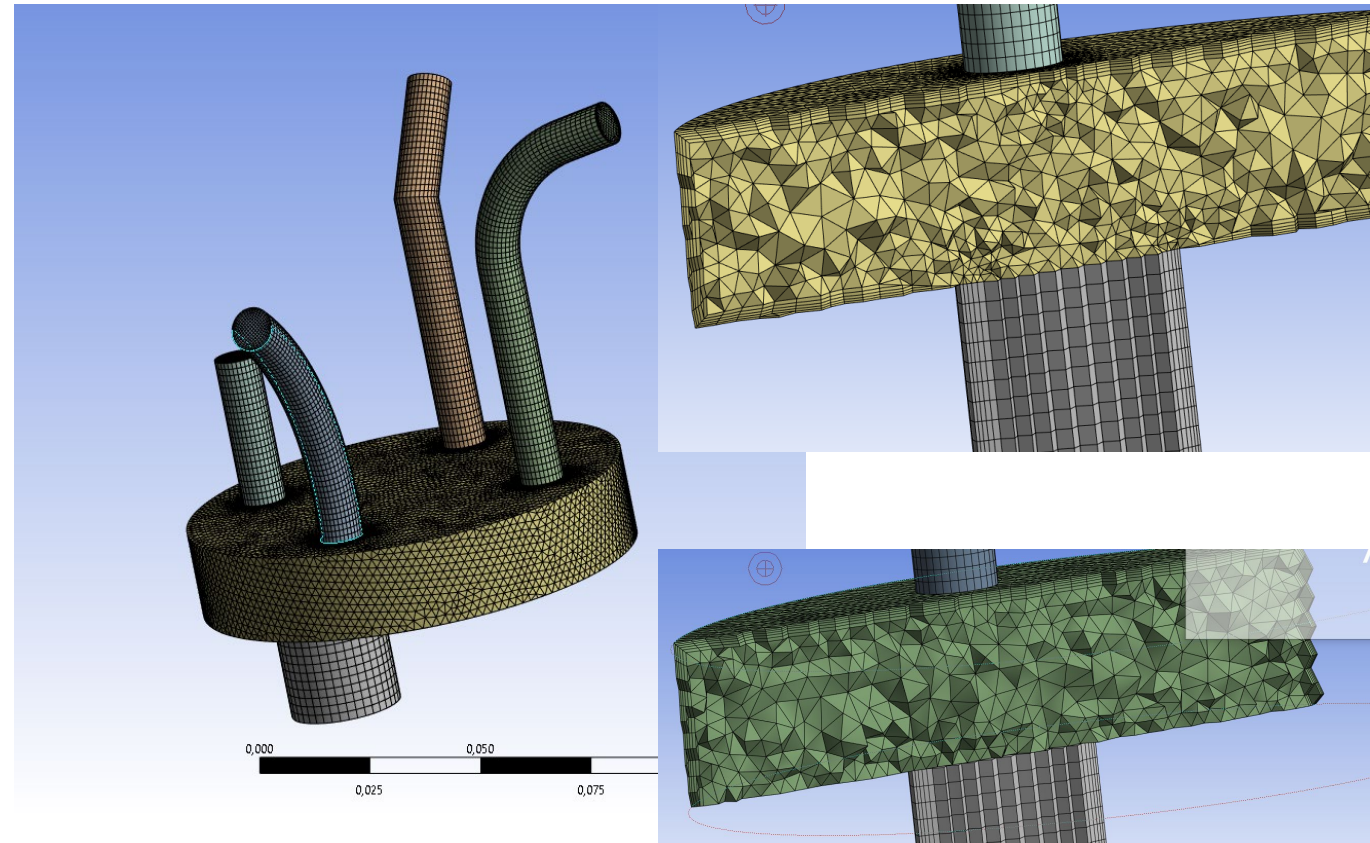
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Unstructured mesh

- The cells arbitrarily distributed
- No indexation of cells
- Meshing can be highly automatized
- useful for highly complex geometry

“Hybrid” mesh

- Tetrahedral mesh in complex regions
- Hexahedral mesh in simpler regions
- Conformal vs non-conformal

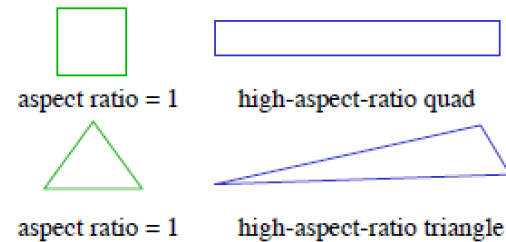


General guidelines

- Hexahedral mesh will give you superior results to tetrahedral mesh and lower cell count with the same cell size
- Mesh should be dense enough to capture flow phenomena
- Local refinements may be used in areas of high gradients
- No mesh is universal it has to be built with application in mind

Quality metrics for shape of cells

- Aspect ratio

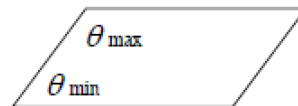


- Skewness

$$\max \left[\frac{\theta_{\max} - \theta_e}{180 - \theta_e}, \frac{\theta_e - \theta_{\min}}{\theta_e} \right]$$

where:

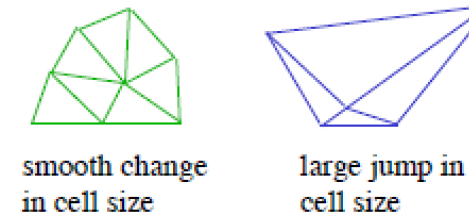
- θ_{\max} = largest angle in face or cell.
- θ_{\min} = smallest angle in face or cell.
- θ_e = angle for equiangular face or cell.
 - e.g., 60 for triangle, 90 for square.



Skewness should not be smaller than 0.85

Quality metrics distribution of cells

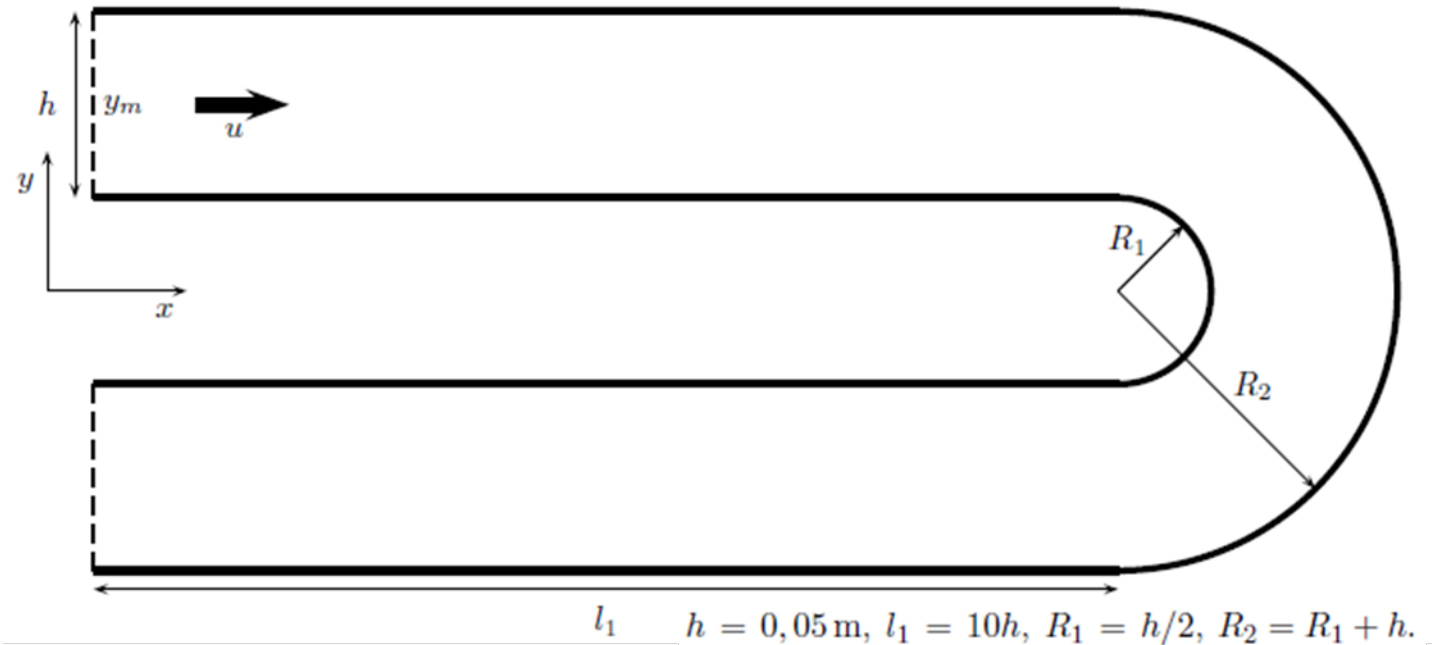
- Smoothness



Sudden jump in cell size should be avoided
Growth ratio should not exceed 1.2

Hands on: 2D u-bend

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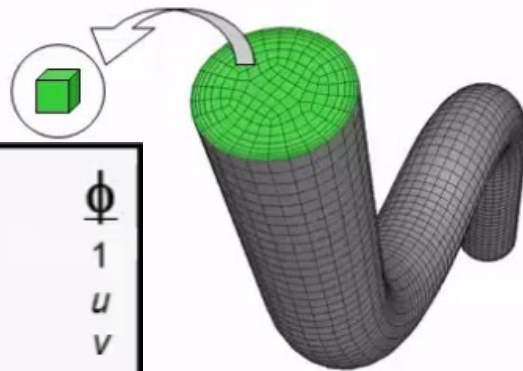
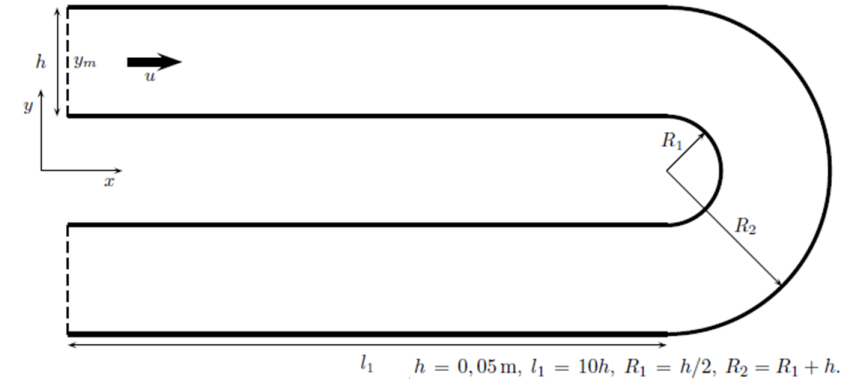


- Steady state, 2D, laminar, isothermal flow of water at standard conditions
 $Re=500; \rho=998.2 \text{ kg/m}^3; \nu=1.003 \times 10^{-6} \text{ m}^2\text{s}^{-1}$
- INLET: Parabolic velocity profile
- OUTLET: Constant ambient pressure

Hands on: 2D u-bend

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$$\underbrace{\frac{\partial}{\partial t} \int_V \rho \phi dV}_{\text{local change}} + \underbrace{\oint_A \rho \phi \mathbf{V} \cdot d\mathbf{A}}_{\text{net flux}} = \underbrace{\oint_A \Gamma_\phi \nabla \phi \cdot d\mathbf{A}}_{\text{net source}} + \underbrace{\int_V S_\phi dV}_{\text{total source}}$$



<u>Equation</u>	ϕ
Continuity	1
X momentum	u
Y momentum	v
Z momentum	w
Energy	h

- How many equation do we have to solve (which variables)
- Which terms can be neglected
- Any special consideration for meshing?

End of Session 1
Thank you for your attention!

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